## SIDDHARTH INSTITUTE OF ENGINEERING \& TECHNOLOGY:: PUTTUR (AUTONOMOUS)

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## OUESTION BANK (DESCRIPTIVE)

Subject with Code: Mechanics of Solids(20CE0164)
Course \& Branch: B.Tech (ME\&AE)
Year \&Sem: II \& I
Regulation: R20

## UNIT I

(Simple Stresses and Strains, Theories of failure)

| 1 | a. Define stress and strain and explain their types. | L1 | CO1 | 6M |
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|  | b. Draw and explain Stress-strain curve for a mild steel bar. | L1 | CO1 | 6M |
| 2 | a. State Hooke's law with equation. | L1 | CO1 | 2M |
|  | b. A tensile test was conducted on a mild steel bar. The following data was obtained from the test : <br> (i) Diameter of the steel bar $=3 \mathrm{~cm}$ <br> (ii) Gauge length of the bar $=20 \mathrm{~cm}$ <br> (iii) Load at elastic limit $=250 \mathrm{KN}$ <br> (iv) Extension at a load of $150 \mathrm{KN}=0.21 \mathrm{~mm}$ <br> (v) Maximum load $=380 \mathrm{KN}$ <br> (vi) Total extension $=60 \mathrm{~mm}$ <br> (vii) Diameter of the rod at the failure $=2.25 \mathrm{~cm}$. <br> Determine : <br> (a) The Young's modulus, <br> (b) The stress at elastic limit, <br> (c) The percentage elongation, and <br> (d) The percentage decrease in area. | L3 | CO1 | 10M |
| 3 | A brass bar, having cross-sectional area of $1000 \mathrm{~mm}^{2}$, is subjected to axial forces as shown in figure. Find the total elongation of the bar.Take $\mathrm{E}=1.05 \times 105 \mathrm{~N} / \mathrm{mm}^{2}$. | L3 | CO1 | 12M |
| 4 | Two brass rods and one steel rod together support a load as shown in fig. If the stresses in brass and steel are not to exceed $60 \mathrm{~N} / \mathrm{mm}^{2}$ and $120 \mathrm{~N} /$ $\mathrm{mm}^{2}$, find the safe load that can be supported. Take E for steel $=2 \times 105$ $\mathrm{N} / \mathrm{mm}^{2}$ and for brass $=1 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$. The cross-sectional area of steel rod is $1500 \mathrm{~mm}^{2}$ and of each brass rod is $1000 \mathrm{~mm}^{2}$ | L3 | CO1 | 12M |



(Shear Force and Bending Moments, Theory of Simple Bending)

| 1 | A cantilever beam of length 3 m carries a uniformly distributed load of 1.5 $\mathrm{KN} / \mathrm{m}$ run over a length of 2 m from the free end. Draw SFD and BMD for the beam. | L3 | CO 2 | 12M |
| :---: | :---: | :---: | :---: | :---: |
| 2 | Draw the shear force and bending moment diagram for a simply supported beam of length 9 m and carrying a uniformly distributed load of $10 \mathrm{KN} / \mathrm{m}$ for a distance of 6 m from the left end. Also calculate the maximum bending moment in the section. | L3 | CO 2 | 12M |
| 3 | Simply supported beam of length 5 m carries a uniformly increasing load of $800 \mathrm{~N} / \mathrm{m}$ at one end to $1600 \mathrm{~N} / \mathrm{m}$ run at the other end. Draw SFD and BMD for the beam. And also calculate the position and magnitude of maximum bending moment. | L3 | CO 2 | 12M |
| 4 | Draw the shear force and bending moment diagram for overhanging beam carrying uniformly distributed load of $2 \mathrm{KN} / \mathrm{m}$ over the entire length and a point load of 2 KN as shown in figure. Locate the point of contra flexure. | L3 | CO 2 | 12M |
| 5 | A simply supported beam oflength10m carries the UDL and two point loads as shown in fig. Draw S.F. and B.M. diagram for the beam shown in figure. Also calculate the maximum bending moment. | L3 | CO 2 | 12M |
| 6 | a) Derive the simple bending equation. | L2 | CO3 | 6M |
|  | b) A beam is simply supported and carries a uniformly distributed load of $40 \mathrm{KN} / \mathrm{m}$ run over the whole span. The section of the hewn is rectangular having depth as 500 mm . If the maximum stress in the material of the beam is $120 \mathrm{~N} / \mathrm{mm}^{2}$ and moment of inertia of the section is $7 \times 10^{8} \mathrm{~mm}^{4}$, find the span of the beam. | L3 | CO3 | 6M |
| 7 | a)State the assumptions made in the theory of simple bending. | L2 | CO3 | 6M |
|  | b) A square beam $20 \mathrm{~mm} \times 20 \mathrm{~mm}$ in section and 2 m long is supported at the ends. The beam fails when a point load of 400 N is applied at the centre of the beam. What uniformly distributed load per metre length will break a cantilever of the same material 40 mm wide, 60 mm deep and 3 m long? | L3 | CO3 | 6M |


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| 8 | a) Derive section modulus for rectangular section. | L2 | CO3 | 4M |
|  | b) A beam 500 mm deep of a symmetrical section has $\mathrm{I}=1 \times 10^{8} \mathrm{~mm}^{4}$ and is simply supported over a span of 10 m . Calculate: <br> (i) The uniformly distributed load it may carry if the maximum bending stress is not to exceed $150 \mathrm{~N} / \mathrm{mm}^{2}$. <br> ii) The bending stress if the beam carries a central point load of 25 KN . | L3 | CO3 | 8M |
| 9 | A cast iron beam is of T-section as shown in figure. The beam is simply supported on a span of 8 m . The beam carries a UDL of $1.5 \mathrm{KN} / \mathrm{m}$ length on the entire span. Determine the maximum tensile and compressive stresses. <br> Fig. | L3 | CO3 | 12M |
| 10 | A cast iron beam is of I-section as shown in figure. The beam is simply supported on a span of 5 m . If the tensile stress is not to exceed $20 \mathrm{~N} / \mathrm{mm}^{2}$, find the safe uniform load which the beam can carry. Find also the maximum compressive stress. | L3 | CO3 | 12M |

## UNIT III

(Shear Stress Distribution, Torsion of Circular Shafts and Springs)

| 1 | a) Derive shear stress distribution formula for rectangular section with a neat sketch. | L1 | CO3 | 6M |
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|  | b)A timber beam of rectangular section is simply supported at the ends and carries a point load at the centre of the beam. The maximum bending stress is $12 \mathrm{~N} / \mathrm{mm}^{2}$ and maximum shearing stress is $1 \mathrm{~N} / \mathrm{mm}^{2}$, find the ratio of the span to the depth. | L3 | CO3 | 6M |
| 2 | a) Derive shear stress distribution formula for circular section with a neat sketch. | L1 | CO3 | 6M |
|  | b) A circular beam of 100 mm diameter is subjected to a shear force of 5 KN . Calculate: <br> (i) Average shear stress <br> (ii) Maximum shear stress <br> (iii) Shear stress at a distance of 40 mm from N.A. | L3 | CO3 | 6M |
| 3 | Draw the shear stress distribution across: <br> (a) Rectangular section. <br> (b) Triangular section. <br> (c) Circular section. <br> (d) I \& T Sections | L2 | CO3 | 12M |
| 4 | a)The shear force acting on a beam at a section is F . The section of the beam is triangular base B and of an altitude H . The beam is placed with its base horizontal. Find the maximum shear stress and the shear stress at the N.A. | L2 | CO3 | 8M |
|  | b)An I-section beam $350 \mathrm{~mm} \times 150 \mathrm{~mm}$ has a web thickness of 10 mm and a flange thickness of 20 mm . If the shear force acting on the section is 40 KN , find the maximum shear stress developed in the I-section. | L3 | CO3 | 4M |
| 5 | The shear force acting on a section of a beam is 50 KN . The section of the beam is of T-shaped of dimensions $100 \mathrm{~mm} \times 100 \mathrm{~mm} \times 20 \mathrm{~mm}$ as shown in figure. The moment of inertia about the horizontal neutral axis is $314.221 \times 10^{4} \mathrm{~mm}^{4}$. Calculate the shear stress at the neutral axis and at the junction of the web and the flange. | L2 | CO3 | 12M |



## UNIT IV

(Deflection of Beams and Columns)

| 1 | Derive the relation between slope, deflection and radius of curvature. | L2 | CO4 | 12M |
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| 2 | A beam of uniform rectangular section 200 mm wide and 300 mm deep is simply supported at its ends. It carries a uniformly distributed load of $9 \mathrm{KN} / \mathrm{m}$ run over the entire span of 5 m . If the value of E for the beam material is $1 \times 10^{4} \mathrm{~N} / \mathrm{mm}^{2}$, find : <br> (i) The slope at the supports and <br> (ii) Maximum deflection. | L3 | CO4 | 12M |
| 3 | Determine: (i) slope at the left support, (ii) deflection under the load and (iii) maximum deflection of a simply supported beam of length 5 m , which is carrying a point load of 5 KN at a distance of 3 m from the left end. Take $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and $\mathrm{I}=1 \times 10^{8} \mathrm{~mm}^{4}$. | L3 | CO4 | 12M |
| 4 | A cantilever of length 3 in carries two point loads of 2 KN at the free end and 4 KN at a distance of 1 m from the free end. Find the deflection at the free end. Take $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and $\mathrm{I}=10^{8} \mathrm{~mm}^{4}$ | L3 | CO4 | 12M |
| 5 | A horizontal beam AB is simply supported at A and B, 6 m apart. The beam is subjected to a clockwise couple of 300 KNm at a distance of 4 m from the left end as shown in figure below If $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and $\mathrm{I}=2 \times 10^{8} \mathrm{~mm}^{4}$, determine : <br> (i) Deflection at the point where couple is acting and <br> (ii) The maximum deflection. | L3 | CO4 | 12M |
| 6 | (a) Write the assumptions made in the Euler's column theory. <br> (b) Write the end conditions for long columns and state the difference between long columns and short columns. | L2 | $\begin{aligned} & \mathrm{CO5} \\ & \mathrm{CO5} \end{aligned}$ | $\begin{aligned} & 4 \mathrm{M} \\ & 8 \mathrm{M} \end{aligned}$ |
| 7 | Derive an expression for crippling load when both ends of the column are hinged. | L2 | CO5 | 12M |
| 8 | A solid round bar 3 m long and 5 cm in diameter is used as a strut with both ends hinged. (Take $\mathrm{E}=2.0 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ ) <br> Determine the crippling load, when the given strut is used with the | L3 | CO5 | 12M |


|  | following conditions : <br> (i) <br> (ii) <br> One end of the strut is fixed and the other end is free <br> (iii) <br> One the ends of strut are fixed |  |  |  |
| :--- | :--- | :--- | :--- | :--- |
| 9 | A column of timber section $15 \mathrm{~cm} \times 20 \mathrm{~cm}$ is 6 metre long both ends <br> being fixed. If the Young's modulus for timber $=17.5 \mathrm{KN} / \mathrm{mm}^{2}$, <br> determine : <br> (i) <br> (ii) <br> (rippling load and <br> Safe load for the column if factor of safety $=3$. | L3 | CO5 | 12 M |
| 10 | Using Euler's formula, calculate the critical stresses for a series of <br> struts having slenderness ratio of $40,80,120,160$ and 200 under the <br> following conditions: <br> (i) Both ends hinged and <br> (ii) Both ends fixed. Take $\mathrm{E}=2.05 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ | L3 | CO5 | 12 M |

# UNIT V <br> (Thin Cylinders and Thick Cylinders) 

| 1 | a) Derive expression for circumferential stress in thin cylinder. <br> b) A cylindrical pipe of diameter 1.5 m and thickness 1.5 cm is subjected to an internal fluid pressure of $1.2 \mathrm{~N} / \mathrm{mm}^{2}$. Determine: <br> i) Longitudinal stress developed in the pipe, and <br> ii) Circumferential stress developed in the pipe. | $\begin{aligned} & \hline \text { L2 } \\ & \text { L3 } \end{aligned}$ | $\begin{array}{\|l\|} \hline \text { CO6 } \\ \text { CO6 } \end{array}$ | $\begin{aligned} & 6 \mathrm{M} \\ & 6 \mathrm{M} \end{aligned}$ |
| :---: | :---: | :---: | :---: | :---: |
| 2 | A cylindrical thin drum 80 cm in diameter and 3 m long has a shell thickness of 1 cm . If the drum is subjected to an internal pressure of $2.5 \mathrm{~N} / \mathrm{mm}^{2}$, Take $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and Poisson's ratio 0.25 <br> Determine <br> (i) change in diameter <br> (ii) change in length and <br> (iii) Change in volume. | L3 | CO6 | 12M |
| 3 | A cylindrical shell 100 mm long 200 mm internal diameter having thickness of a metal as 10 mm is filled with a fluid at atmospheric pressure. If an additional $200 \mathrm{~mm}^{3}$ pumped into the cylinder, Take $\mathrm{E}=$ $2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and Poisson's ratio is 0.3 . Find <br> (i) The pressure exerted by the fluid on the cylinder and <br> (ii) The hoop stress induced. | L3 | CO6 | 12M |
| 4 | A copper cylinder, 90 cm long, 40 cm external diameter and wall thickness 6 mm has its both ends closed by rigid blank flanges. It is initially full of oil at atmospheric pressure. Calculate additional volume of oil which must be pumped into it in order to raise the oil pressure to $5 \mathrm{~N} / \mathrm{mm}^{2}$ above atmospheric pressure. For copper assume $\mathrm{E}=1.0 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and Poisson's ratio $1 / 3$. Take bulk modulus of oil as $\mathrm{K}=2.6 \times 10^{3} \mathrm{~N} / \mathrm{mm}^{2}$. | L3 | CO6 | 12M |
| 5 | A closed cylindrical vessel made of steel plates 4 mm thick with plane and, carries fluid under a pressure of $3 \mathrm{~N} / \mathrm{mm}^{2}$. The dia. of cylinder is 30 cm and length is 80 cm , calculate the longitudinal and hoop stresses in the cylinder wall and determine the change in diameter, length and volume of the cylinder. Take $\mathrm{E}=2 \mathrm{X} 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$ and Poisson's ratio is 0.286 | L3 | CO6 | 12M |
| 6 | a) A cylinder of thickness 1.5 cm has to withstand maximum internal pressure of $1.5 \mathrm{~N} / \mathrm{mm}^{2}$. If the ultimate tensile stress in the material of the cylinder is $300 \mathrm{~N} / \mathrm{mm}^{2}$, factor of safety 3.0 and joint efficiency $80 \%$, determine the diameter of the cylinder. <br> b) A spherical shell of internal diameter 0.9 m and of thickness 10 mm is subjected to an internal pressure of $1.4 \mathrm{~N} / \mathrm{mm}^{2}$. Determine the increase in diameter and increase in volume. Take $\mathrm{E}=2 \mathrm{X} 10^{5}$ $\mathrm{N} / \mathrm{mm}^{2}$ and $\mu=1 / 3$. | L3 | CO6 CO6 | 6 M 6 M |
| 7 | Derive an expression for hoop and radial stresses across thickness of the thick cylinder. | L2 | CO6 | 12M |
| 8 | Determine the maximum and minimum hoop stress across the section of a pipe of 400 mm internal diameter and 100 mm thick, when the pipe contains a fluid at a pressure of $8 \mathrm{~N} / \mathrm{mm}^{2}$. Also sketch the radial | L3 | CO6 | 12M |


|  | pressure and hoop stress distribution across the section. |  |  |  |
| :--- | :--- | :--- | :--- | :--- |
| 9 | A compound cylinder is made by shrinking a cylinder of external <br> diameter 300 mm and internal diameter of 250 mm over another <br> cylinder of external diameter 250 mm and internal diameter 200 mm. <br> The radial pressure at the junction after shrinking is $8 \mathrm{~N} / \mathrm{mm}^{2}$. Find <br> the final stresses set up across the section, when the compound <br> cylinder is subjected to an internal fluid pressure of $84.5 \mathrm{~N} / \mathrm{mm}^{2}$. | CO6 | 12 M |  |
| 10 | A steel cylinder of 300 mm external diameter is to be shrunk to <br> another steel cylinder of 150 mm internal diameter. After shrinking, <br> the diameter at the junction is 250 mm and radial pressure at the <br> common junction is $28 \mathrm{~N} / \mathrm{mm}^{2}$. Find the original difference in radii at <br> the junction. Take $\mathrm{E}=2 \times 10^{5} \mathrm{~N} / \mathrm{mm}^{2}$. | CO6 | 12 M |  |

